



# Numerical Integration

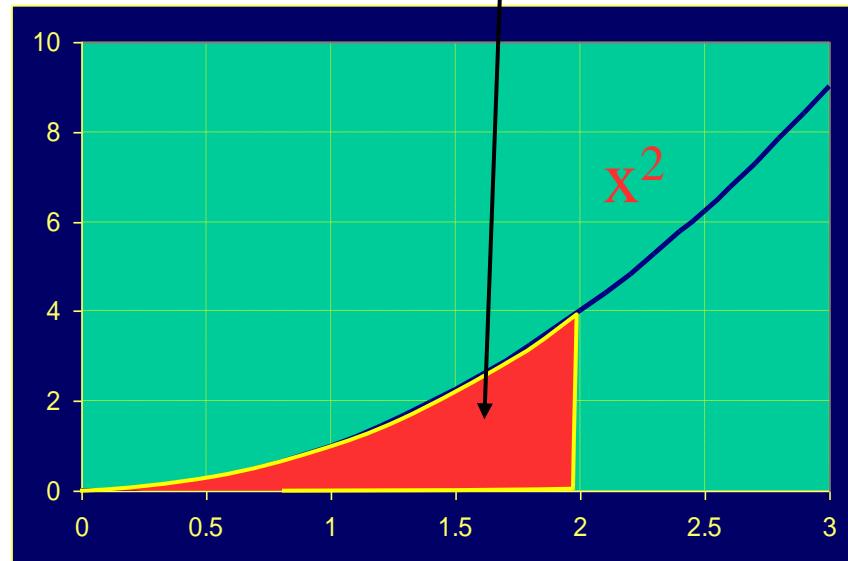
## Integrals

Indefinite

$$\int x^2 dx = \frac{1}{3}x^3 + C$$

Definite

$$\int_0^2 x^2 dx = \frac{8}{3}$$

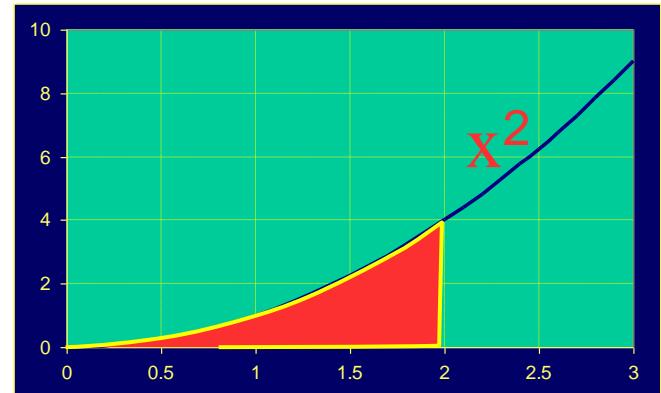


# Numerical Integration

- Can be solved exactly, but for various reasons FEA prefers to evaluate integrals like this approximately:
  - Historically, considered more efficient and reduced coding errors.
  - Only possible approach for isoparametric elements.
  - Can actually improve performance in certain cases!
- Definite integrals can be computed numerically

$$\int_a^b f(x)dx \cong \sum_i w_i f(x_i)$$

- Objective:
  - Determine points  $x_i$
  - Determine coefficients  $w_i$





# Numerical Integration

➤ Depending on choice of  $w_i$  and  $x_i$

- Midpoint Rule
- Trapezoidal Rule
- Simpson's
- Gaussian Quadratures
- etc

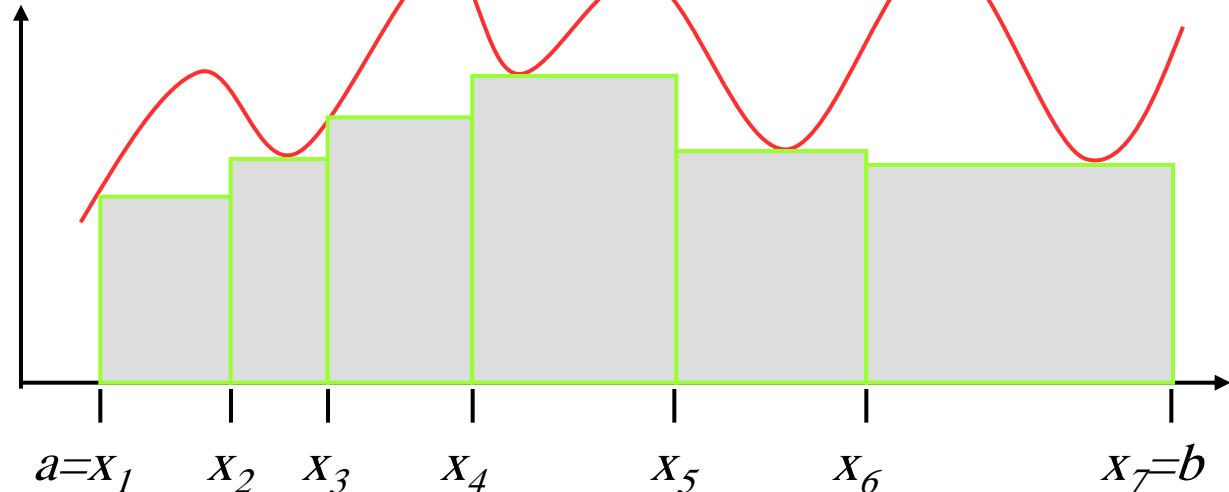
# Numerical Integration

## Numerical Integration – Upper & Lower Bounds

Depending on  
choice of  $w_i$  and  $x_i$

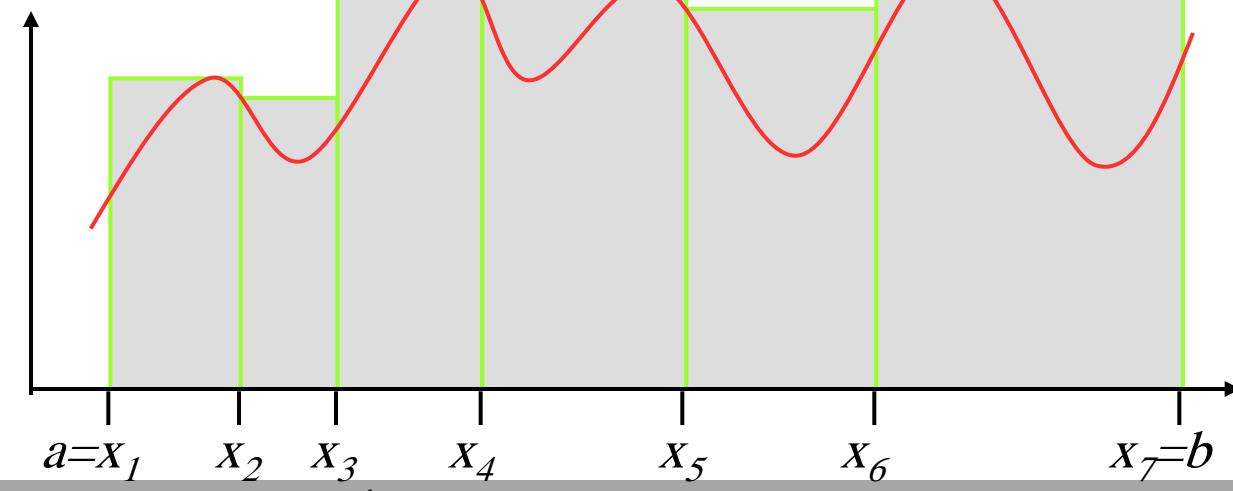
Lower Sum

$$L(f; X_i)$$



Upper Sum

$$U(f; X_i)$$





# Numerical Integration

It can be shown that

$$L(f; x_i) \leq \int_a^b f(x) dx \cong \sum_i w_i f(x_i) \leq U(f; x_i)$$

$$\lim_{i \rightarrow \infty} L(f; x_i) = \int_a^b f(x) dx = \sum_i w_i f(x_i) = \lim_{i \rightarrow \infty} U(f; x_i)$$

## Objective

$$\int_a^b f(x) dx \cong \sum_i w_i f(x_i) = w_1 f(x_1) + w_2 f(x_2) + \cdots + w_n f(x_n)$$

Where do such formulae come from?

Theory of Interpolation....

Let  $f(x) \approx p(x) = \sum_{i=1}^n l_i(x) f(x_i)$   $l_i(x)$ : cardinal functions

Recall Shape Functions



# Numerical Integration: Quadratures

$$\int_a^b f(x)dx \approx \int_a^b p(x)dx = \sum_{i=1}^n f(x_i) \int_a^b l_i(x)dx = \sum_{i=1}^n f(x_i)w_i$$

It will give correct values for the integral of every polynomial of degree  $\leq n-1$

## Gaussian Quadrature:

Karl Friedreich Gauss discovered that by a special placement of nodes the accuracy of the numerical integration could be greatly increased



# Numerical Integration: Gaussian Quadrature

Theorem on Gaussian nodes

Let  $q$  be a polynomial of degree  $n$  such that

$$\int_a^b q(x) x^k dx = 0 \quad k = 0, 1, \dots, n-1$$

Let  $x_1, x_2, \dots, x_n$  be the roots of  $q(x)$ . Then

$$\int_a^b f(x) dx \cong \sum_i w_i f(x_i) = w_1 f(x_1) + w_2 f(x_2) + \dots + w_n f(x_n)$$

with  $x_i$ 's as nodes is exact for all polynomials of degree  $\leq 2n-1$ .



# Numerical Integration: Gaussian Quadrature

Assume two point formulation, then:

$$\int_{-1}^1 F(\xi) d\xi = w_1 F(\xi_1) + w_2 F(\xi_2)$$

Four equations are created using Legendre polynomials  $(1, \xi, \xi^2, \xi^3)$

$$w_1 F(\xi_1) + w_2 F(\xi_2) = \int_{-1}^1 1 d\xi = 2$$

$$w_1(I) + w_2(I) = 2$$

$$w_1 F(\xi_1) + w_2 F(\xi_2) = \int_{-1}^1 \xi d\xi = 0$$

$$w_1(\xi_1) + w_2(\xi_2) = 0$$

$$w_1 F(\xi_1) + w_2 F(\xi_2) = \int_{-1}^1 \xi^2 d\xi = 2/3$$

$$w_1(\xi_1)^2 + w_2(\xi_2)^2 = 2/3$$

$$w_1 F(\xi_1) + w_2 F(\xi_2) = \int_{-1}^1 \xi^3 d\xi = 0$$

$$w_1(\xi_1)^3 + w_2(\xi_2)^3 = 0$$

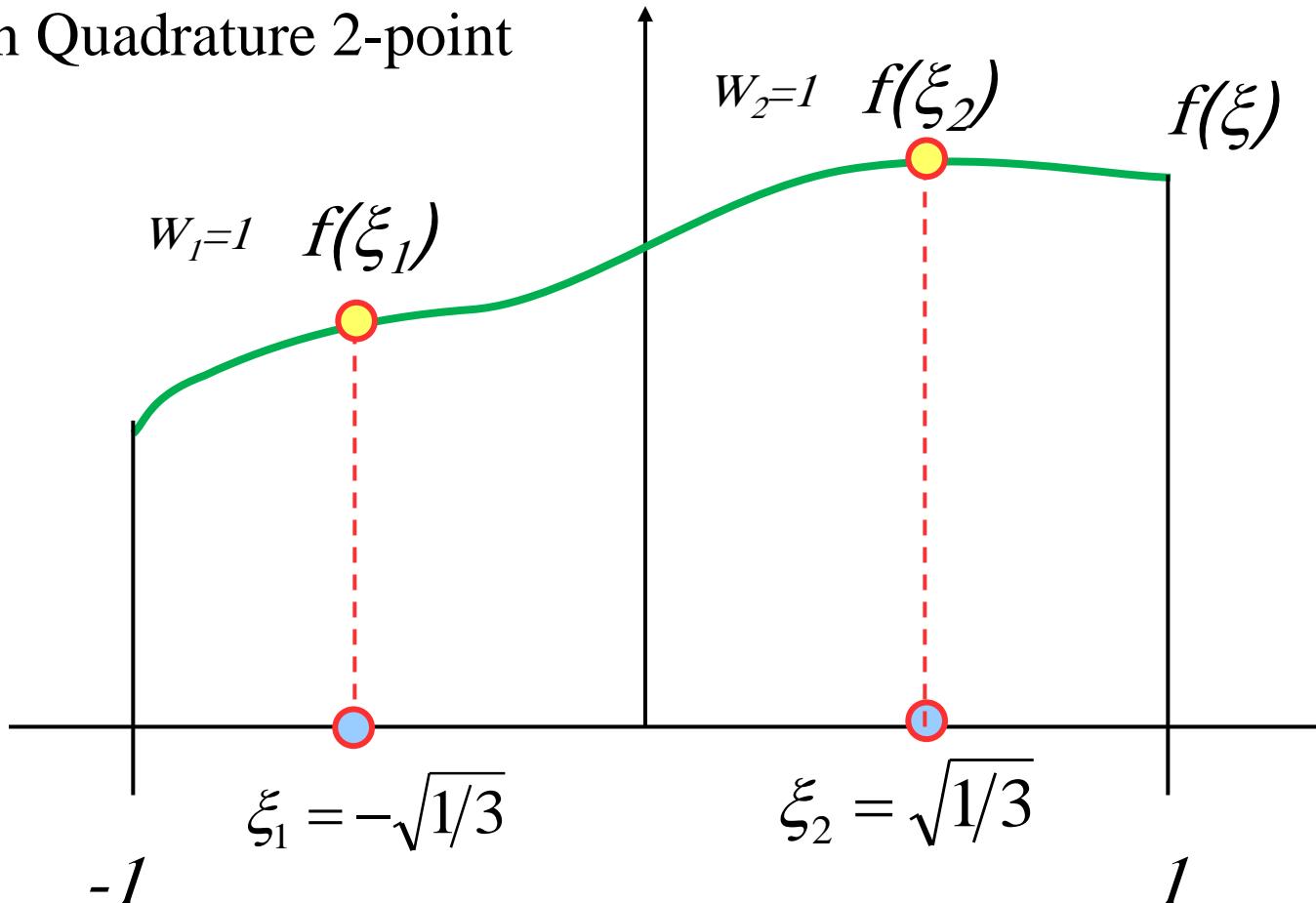
$$w_1 = w_2 = 1$$

$$\xi_1 = -1/\sqrt{3}$$

$$\xi_2 = 1/\sqrt{3}$$

# Numerical Integration: Gaussian Quadrature

Gaussian Quadrature 2-point



$$\int_{-1}^1 f(x) dx \cong 1 * f(-\sqrt{1/3}) + 1 * f(\sqrt{1/3})$$

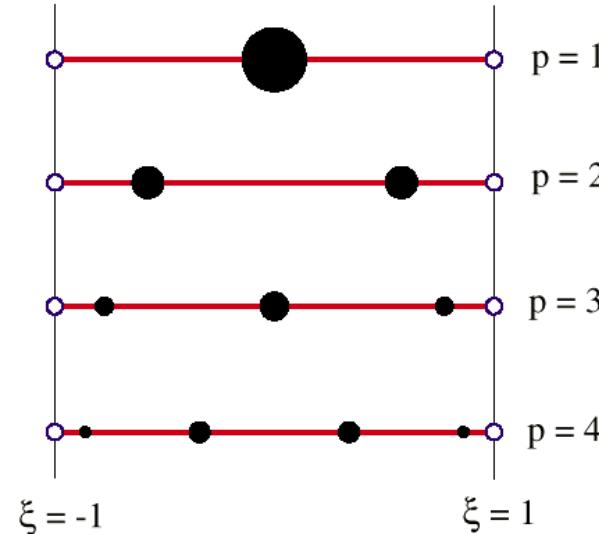
# Numerical Integration: Gaussian Quadrature

## One Dimensional Gauss Integration Rules:

One point:  $\int_{-1}^1 F(\xi) d\xi \doteq 2F(0),$

Two points:  $\int_{-1}^1 F(\xi) d\xi \doteq F(-1/\sqrt{3}) + F(1/\sqrt{3}),$

Three points:  $\int_{-1}^1 F(\xi) d\xi \doteq \frac{5}{9}F(-\sqrt{3/5}) + \frac{8}{9}F(0) + \frac{5}{9}F(\sqrt{3/5})$





# Numerical Integration: Gaussian Quadrature

## Weighting Factors & Sampling Points for Gauss-Legendre Formula

Points( $n$ )	Weighting Factor ( $w_i$ )	Sampling Points ( $\xi_i$ )
2	$w_1 = 1.000000000$	$\xi_1 = -.577350269$
	$w_2 = 1.000000000$	$\xi_2 = .577350269$
3	$w_1 = 0.555555556$	$\xi_1 = -.774596669$
	$w_2 = 0.888888889$	$\xi_2 = 0.0$
	$w_3 = 0.555555556$	$\xi_3 = 0.774596669$
4	$w_1 = 0.3478548$	$\xi_1 = -.861136312$
	$w_2 = 0.6521452$	$\xi_2 = -.339981044$
	$w_3 = 0.6521452$	$\xi_3 = 0.339981044$
	$w_4 = 0.3478548$	$\xi_4 = .861136312$
5	$w_1 = 0.2369269$	$\xi_1 = -.906179846$
	$w_2 = 0.4786287$	$\xi_2 = -.538469310$
	$w_3 = 0.5688889$	$\xi_3 = 0.0$
	$w_4 = 0.4786287$	$\xi_4 = .538469310$
	$w_5 = 0.2369269$	$\xi_5 = .906179846$



# Numerical Integration: Gaussian Quadrature

Example:

$$I = \int_{-2}^{6} (x^2 + 5x + 3) dx = ? \quad \text{Analytical solution} \rightarrow 161.3333$$

$$x = 4 + 2\xi \quad \rightarrow \quad I = \int_{-1}^{1} 2[(4 + 2\xi)^2 + 5(4 + 2\xi) + 3] d\xi$$

$F(\xi)$

$$I = w_1 F(\xi_1) + w_2 F(\xi_2) = F\left(\frac{1}{\sqrt{3}}\right) + F\left(\frac{-1}{\sqrt{3}}\right)$$

$$I = (1)(50.64445) + (1)(110.68888) = 161.3333$$



# Numerical Integration: Gaussian Quadrature

## Two Dimensional Product Gauss Rules

Canonical form of integral:

$$\int_{-1}^1 \int_{-1}^1 F(\xi, \eta) d\xi d\eta = \int_{-1}^1 d\eta \int_{-1}^1 F(\xi, \eta) d\xi$$

Gauss integration rules with  $p_1$  points in the  $\xi$  direction and  $p_2$  points in the  $\eta$  direction:

$$\int_{-1}^1 \int_{-1}^1 F(\xi, \eta) d\xi d\eta = \int_{-1}^1 d\eta \int_{-1}^1 F(\xi, \eta) d\xi \approx \sum_{i=1}^{p_1} \sum_{j=1}^{p_2} w_i w_j F(\xi_i, \eta_j)$$

Usually  $p1 = p2 = p$

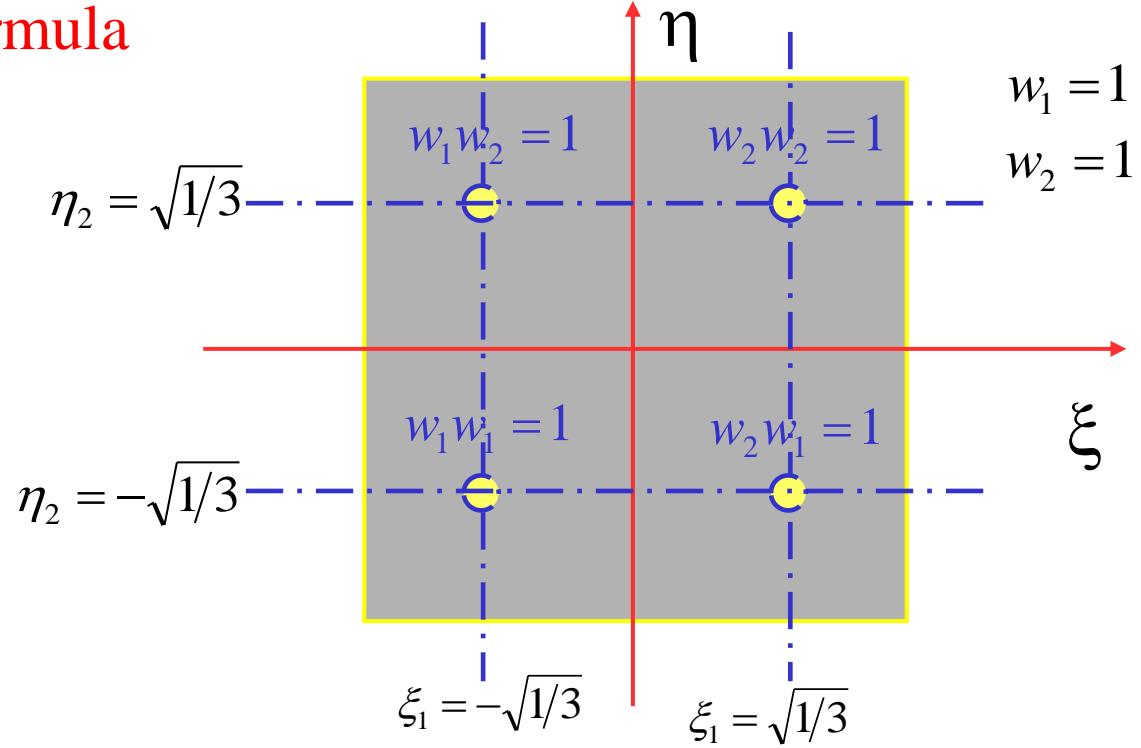
## 2-Dimensional Integration: Gaussian Quadrature

$$\int_{-1}^1 \int_{-1}^1 f(\xi, \eta) d\xi d\eta \cong \int_{-1}^1 \left[ \sum_{i=1}^n w_i f(\xi_i, \eta) \right] d\eta \approx \sum_{j=1}^n \sum_{i=1}^n w_j w_i f(\xi_i, \eta_j)$$

### 2-D Integration 2-point formula

$$\int_{-1}^1 \int_{-1}^1 f(\xi, \eta) d\xi d\eta \cong$$

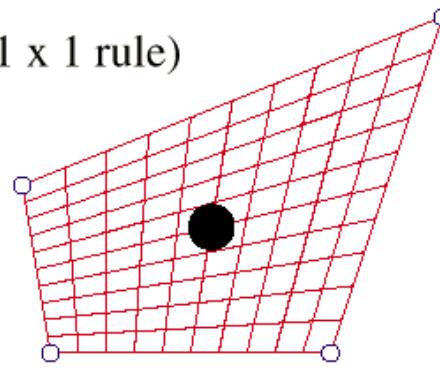
$$w_1 w_1 f(\xi_1, \eta_1) + w_2 w_1 f(\xi_2, \eta_1) + w_1 w_2 f(\xi_1, \eta_2) + w_2 w_2 f(\xi_2, \eta_2)$$



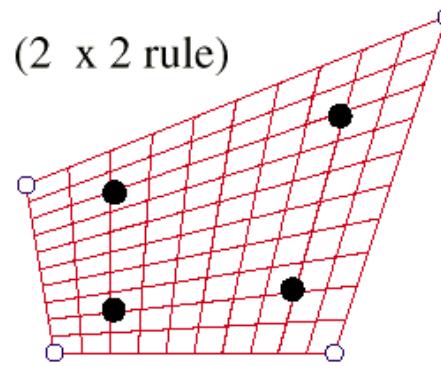
# Numerical Integration: Gaussian Quadrature

Graphical Representation of the First Four 2D Product-Type Gauss Integration Rules

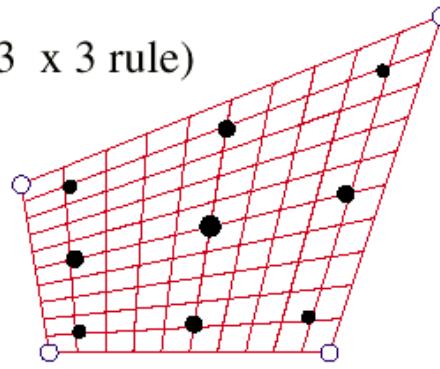
$p = 1$  (1 x 1 rule)



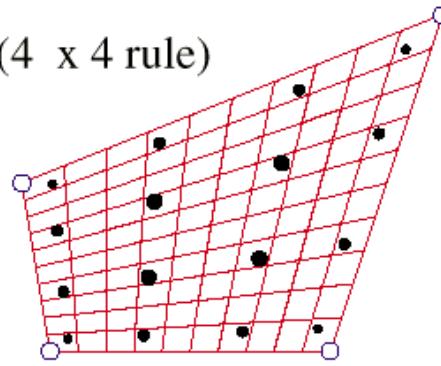
$p = 2$  (2 x 2 rule)



$p = 3$  (3 x 3 rule)



$p = 4$  (4 x 4 rule)



With Equal # of Points  $p$  in Each Direction

## Integration of Stiffness Matrix

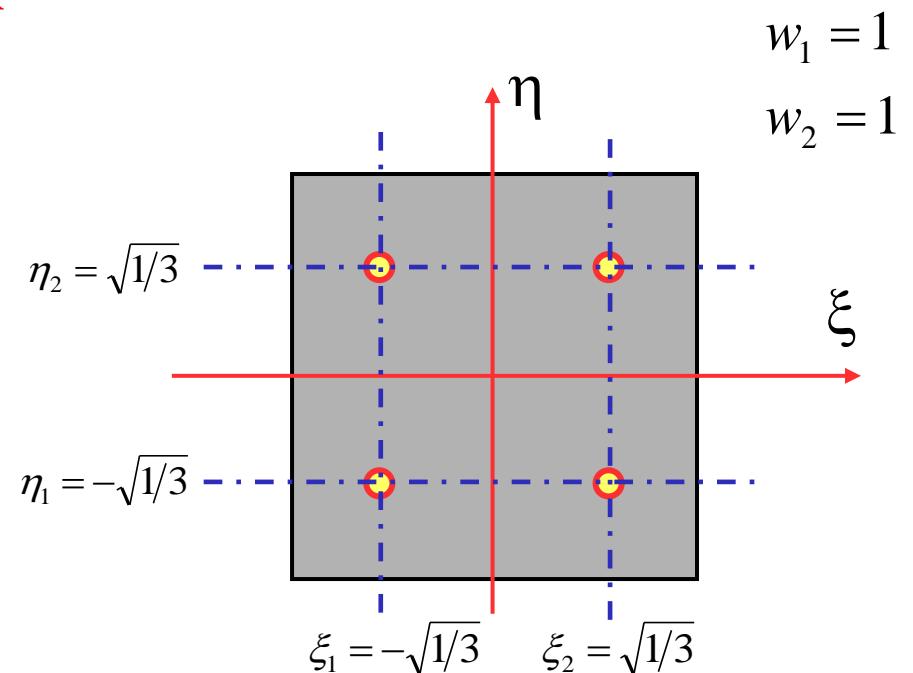
$$\mathbf{k} = t \int_A \mathbf{B}^T \mathbf{D} \mathbf{B} dA$$

$$= t \int_{-1}^1 \int_{-1}^1 \mathbf{B}^T \mathbf{D} \mathbf{B} \det J d\xi d\eta$$

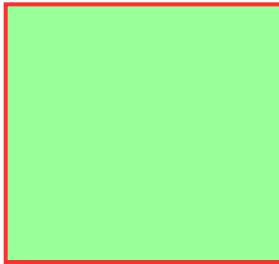
$$k_{ij} = t \int_{-1}^1 \int_{-1}^1 g_{ij}(\xi, \eta) d\xi d\eta$$

$$= w_1 w_1 g_{ij}(\xi_1, \eta_1) + w_2 w_1 g_{ij}(\xi_2, \eta_1) + w_1 w_2 g_{ij}(\xi_1, \eta_2) + w_2 w_2 g_{ij}(\xi_2, \eta_2)$$

$$= g_{ij}(\xi_1, \eta_1) + g_{ij}(\xi_2, \eta_1) + g_{ij}(\xi_1, \eta_2) + g_{ij}(\xi_2, \eta_2)$$



## Modeling Issues: Element Shape



Square : Optimum Shape

Not always possible to use

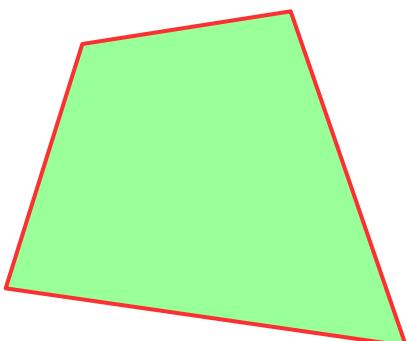


Rectangles:

Rule of Thumb

Ratio of sides  $< 2$

Larger ratios  
may be used  
with caution

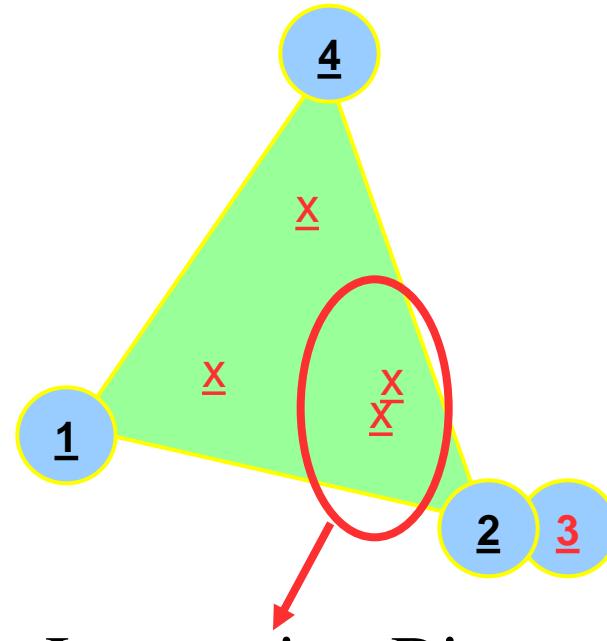
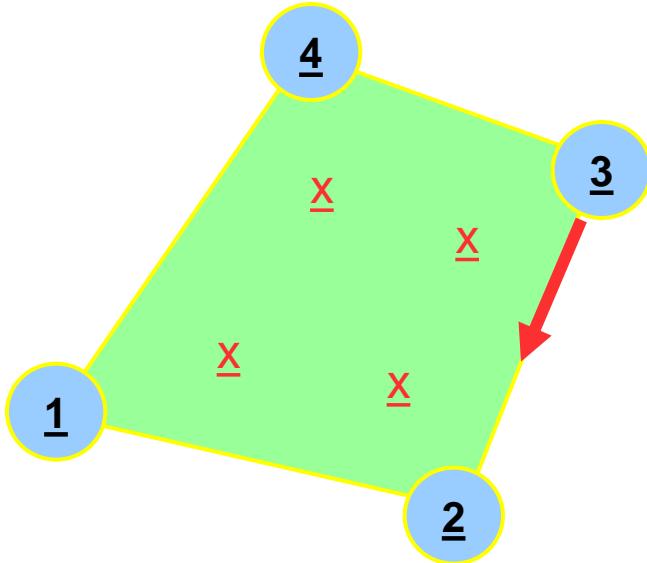


Angular Distortion

Internal Angle  $< 180^\circ$

## Modeling Issues: Degenerate Quadrilaterals

Coincident Corner Nodes



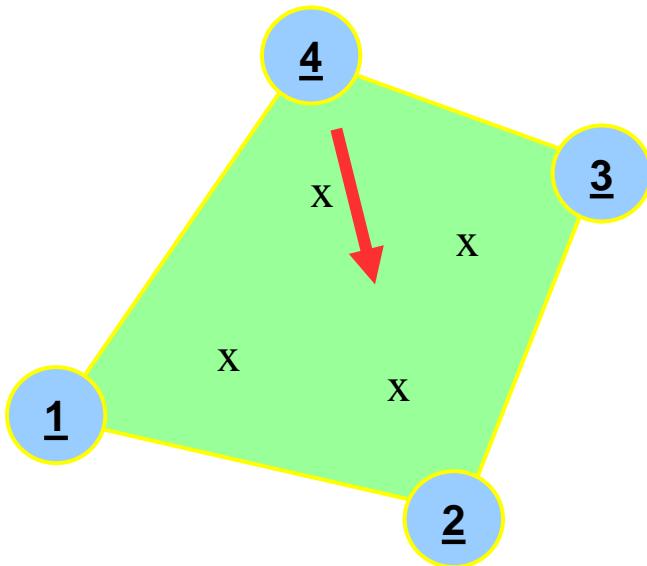
Integration Bias

Less accurate

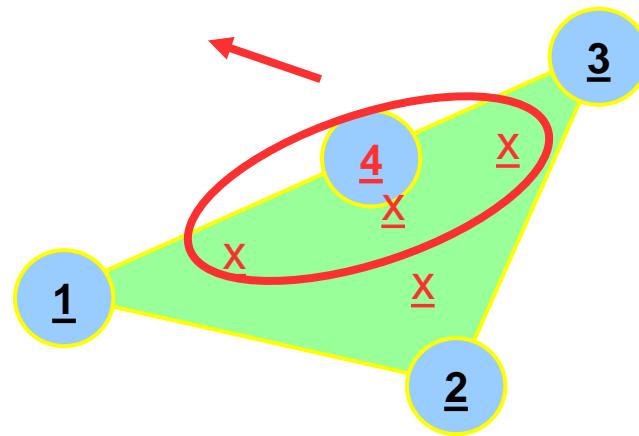
# Numerical Integration: Gaussian Quadrature

## Modeling Issues: Degenerate Quadrilaterals

Three nodes collinear



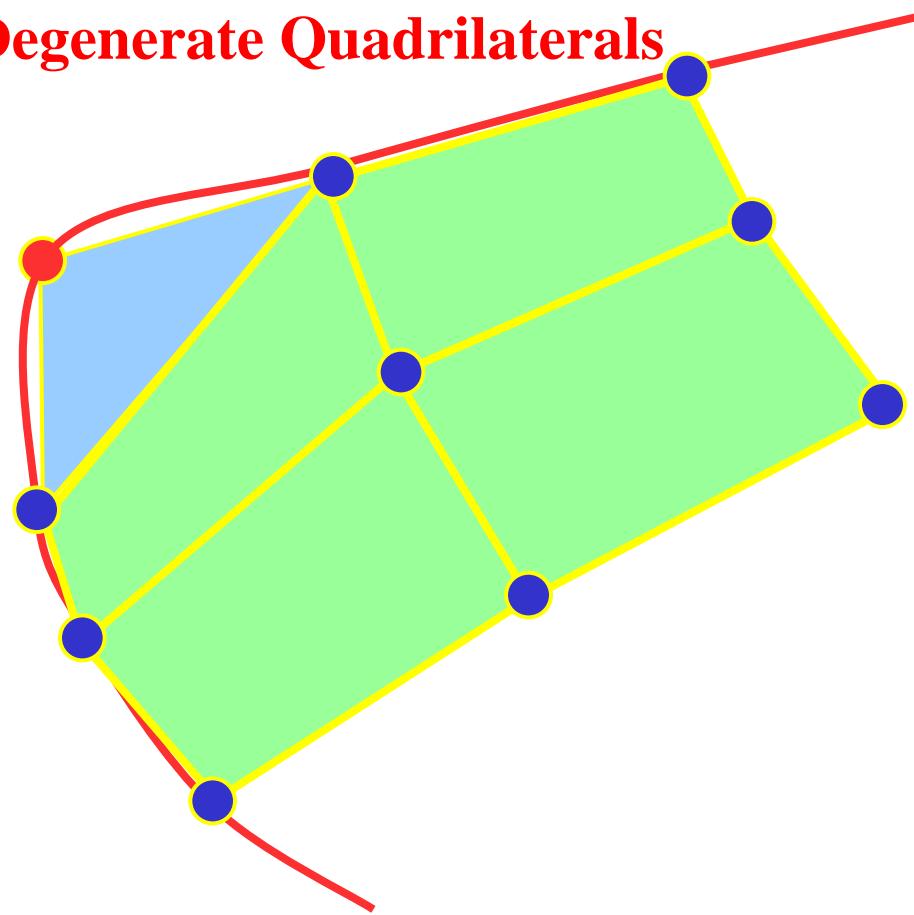
Integration Bias



Less accurate

## Modeling Issues: Degenerate Quadrilaterals

2 nodes

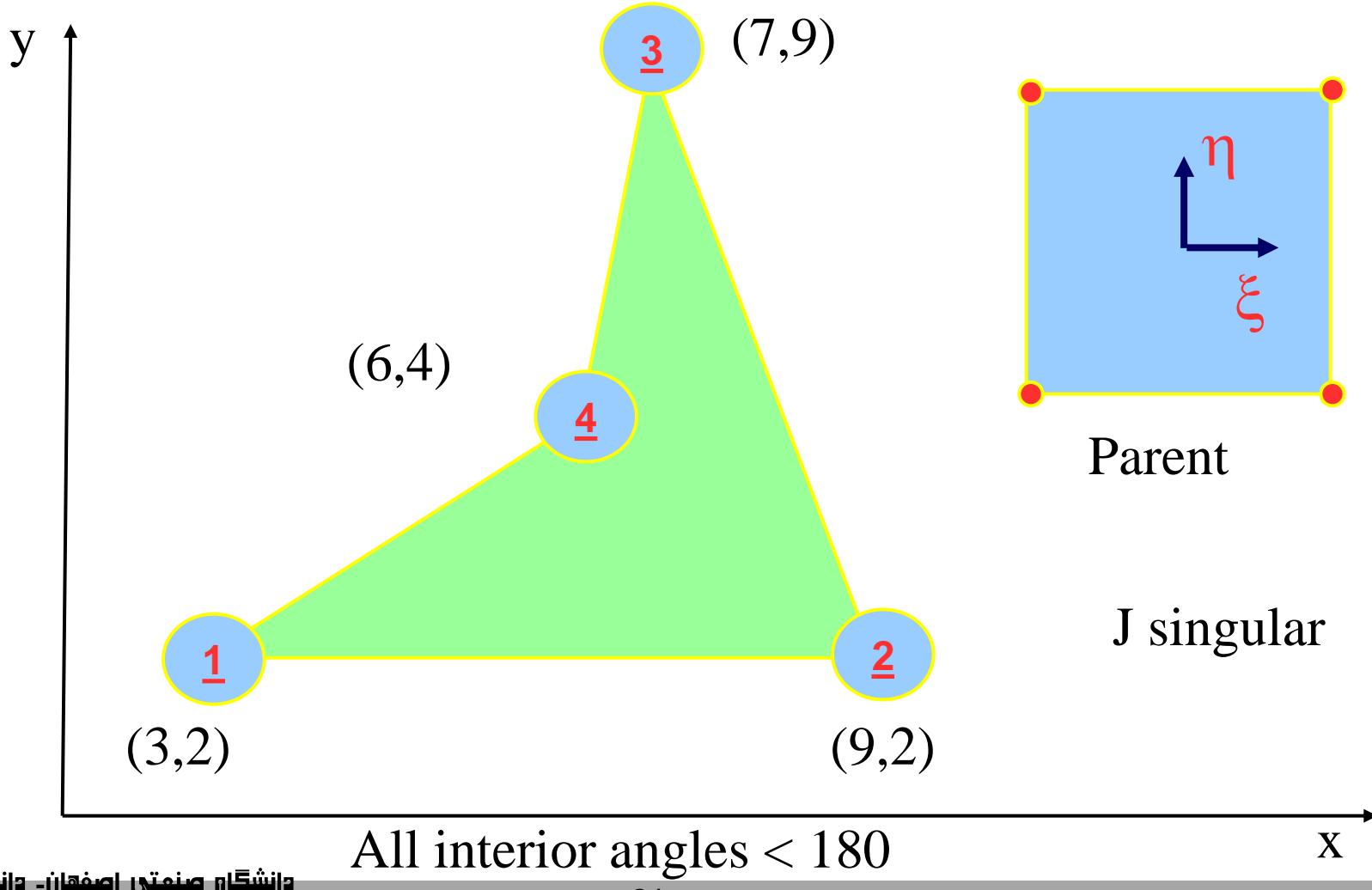


Use only as necessary to improve representation of geometry  
Do not use in place of triangular elements

# Numerical Integration: Gaussian Quadrature

## Modeling Issues: Degenerate Quadrilaterals

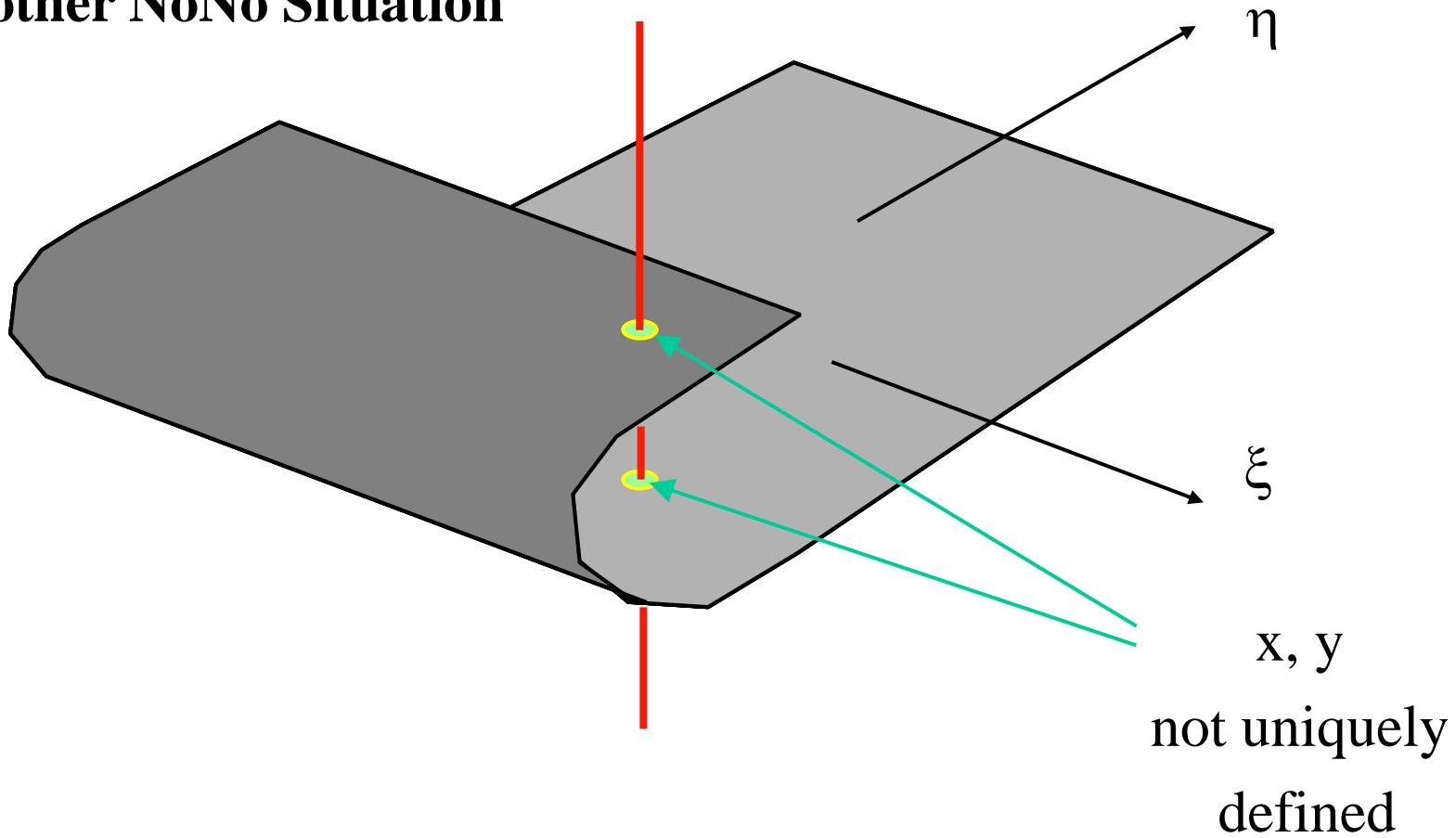
A NoNo Situation



# Numerical Integration: Gaussian Quadrature

## Modeling Issues: Degenerate Quadrilaterals

### Another NoNo Situation



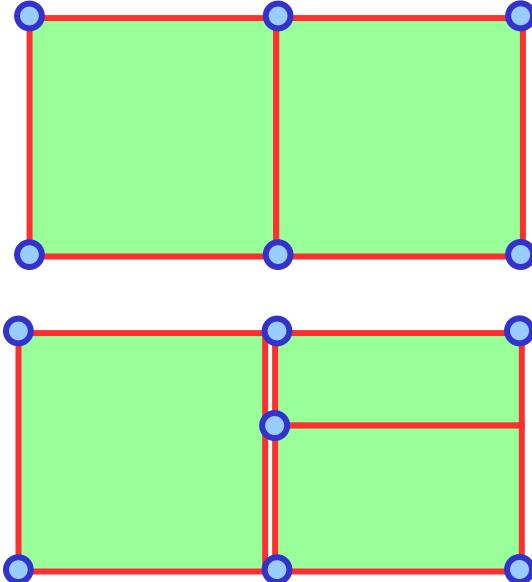
## Convergence Considerations

For monotonic convergence of solution; Requirements

Elements (mesh) must be compatible

Elements must be complete

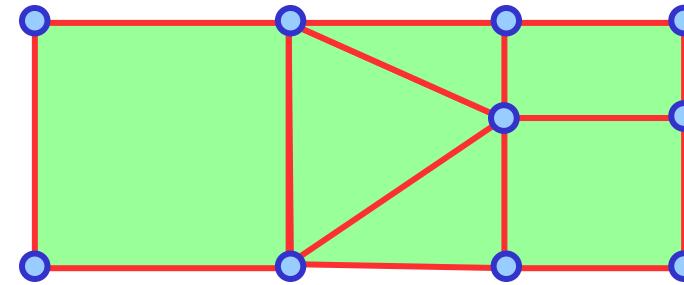
### Mesh Compatibility



OK

NO!

### Mesh compatibility - Refinement



Acceptable Transition

Compatibility of displacements OK  
Stresses?



# Numerical Integration: Gaussian Quadrature

## Gauss integration

- For evaluation of integrals in  $\mathbf{k}$  (in practice)

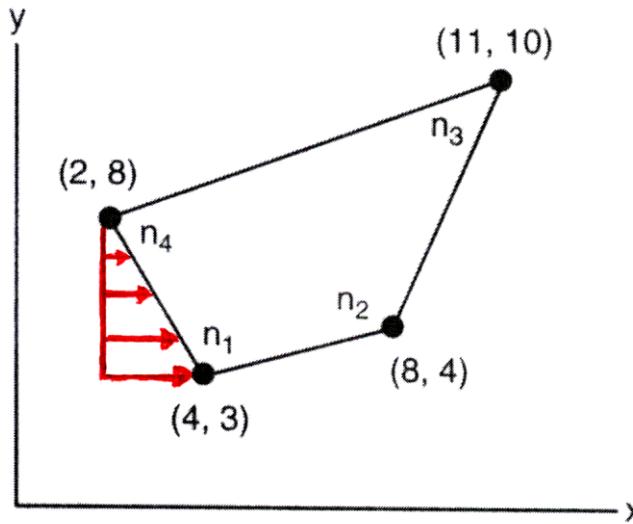
$$\text{In 1 direction: } I = \int_{-1}^{+1} f(\xi) d\xi = \sum_{j=1}^m w_j f(\xi_j)$$

$m$  gauss points gives exact solution of polynomial integrand of  $n = 2m - 1$

$$\text{In 2 directions: } I = \int_{-1}^{+1} \int_{-1}^{+1} f(\xi, \eta) d\xi d\eta = \sum_{i=1}^{n_x} \sum_{j=1}^{n_y} w_i w_j f(\xi_i, \eta_j)$$

# Numerical Integration: Gaussian Quadrature

Example:



$$\mathbf{t}(x, y) = \begin{pmatrix} 0.4 * (8 - y) \\ 0 \end{pmatrix} \text{ ksi}$$

- Given: 4-node plane stress element has  $E = 30,000$  ksi,  $\nu = 0.25$ ,  $h = 0.50$  in, no body force, and surface traction shown.
- Required: Find  $\mathbf{k}$  and  $\mathbf{f}$ . Use  $2 \times 2$  Gauss quadrature for  $\mathbf{k}$ .



# Numerical Integration: Gaussian Quadrature

## Solution:

➤ Isoparametric mapping:

$$\begin{aligned}x &= \frac{1}{4}(1-\xi)(1-\eta)x_1 + \frac{1}{4}(1+\xi)(1-\eta)x_2 + \frac{1}{4}(1+\xi)(1+\eta)x_3 + \frac{1}{4}(1-\xi)(1+\eta)x_4 \\&= \frac{1}{4}(1-\xi)(1-\eta)*4 + \frac{1}{4}(1+\xi)(1-\eta)*8 + \frac{1}{4}(1+\xi)(1+\eta)*11 + \frac{1}{4}(1-\xi)(1+\eta)*2 \\&= \frac{25}{4} + \frac{13}{4}\xi + \frac{1}{4}\eta + \frac{5}{4}\xi\eta; \\y &= \frac{1}{4}(1-\xi)(1-\eta)*3 + \frac{1}{4}(1+\xi)(1-\eta)*4 + \frac{1}{4}(1+\xi)(1+\eta)*10 + \frac{1}{4}(1-\xi)(1+\eta)*8 \\&= \frac{25}{4} + \frac{3}{4}\xi + \frac{11}{4}\eta + \frac{1}{4}\xi\eta;\end{aligned}$$

➤ Jacobian matrix and Jacobian:

$$\mathbf{J} = \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial y}{\partial \xi} \\ \frac{\partial x}{\partial \eta} & \frac{\partial y}{\partial \eta} \end{bmatrix} = \begin{bmatrix} \frac{13}{4} + \frac{5}{4}\eta & \frac{3}{4} + \frac{1}{4}\eta \\ \frac{1}{4} + \frac{5}{4}\xi & \frac{11}{4} + \frac{1}{4}\xi \end{bmatrix}; \quad J = \det \mathbf{J} = \frac{35}{4} - \frac{1}{8}\xi + \frac{27}{8}\eta.$$



# Numerical Integration: Gaussian Quadrature

► B matrix:

$$[B(\xi, \eta)] = \frac{1}{|\mathbf{J}|} [B_1 \mid B_2 \mid B_3 \mid B_4]$$

$$[B_i] = \begin{bmatrix} a(N_{i,\xi}) - b(N_{i,\eta}) & 0 \\ 0 & c(N_{i,\eta}) - d(N_{i,\xi}) \\ c(N_{i,\eta}) - d(N_{i,\xi}) & a(N_{i,\xi}) - b(N_{i,\eta}) \end{bmatrix}$$



# Numerical Integration: Gaussian Quadrature

► B matrix:

$$N_{1,\xi} = \frac{\partial N_1}{\partial \xi} = \frac{-1(1-\eta)}{4} = \frac{(\eta-1)}{4}$$

$$N_{2,\xi} = \frac{\partial N_2}{\partial \xi} = \frac{(1)(1-\eta)}{4} = \frac{(1-\eta)}{4}$$

$$N_{3,\xi} = \frac{\partial N_3}{\partial \xi} = \frac{(1)(1+\eta)}{4} = \frac{(1+\eta)}{4}$$

$$N_{4,\xi} = \frac{\partial N_4}{\partial \xi} = \frac{(-1)(1+\eta)}{4} = \frac{-(1+\eta)}{4}$$

$$N_{1,\eta} = \frac{\partial N_1}{\partial \eta} = \frac{(1-\xi)(-1)}{4} = \frac{(\xi-1)}{4}$$

$$N_{2,\eta} = \frac{\partial N_2}{\partial \eta} = \frac{(1+\xi)(-1)}{4} = \frac{-(\xi+1)}{4}$$

$$N_{3,\eta} = \frac{\partial N_3}{\partial \eta} = \frac{(1+\xi)(1)}{4} = \frac{(\xi+1)}{4}$$

$$N_{4,\eta} = \frac{\partial N_4}{\partial \eta} = \frac{(1-\xi)(1)}{4} = \frac{(1-\xi)}{4}$$

$$a = 1/4 \left[ y_1 (\xi - 1) + y_2 (-\xi - 1) + y_3 (\xi + 1) + y_4 (1 - \xi) \right]$$

$$b = 1/4 \left[ y_1 (\eta - 1) + y_2 (1 - \eta) + y_3 (\eta + 1) + y_4 (-1 - \eta) \right]$$

$$c = 1/4 \left[ x_1 (\eta - 1) + x_2 (1 - \eta) + x_3 (\eta + 1) + x_4 (-1 - \eta) \right]$$

$$d = 1/4 \left[ x_1 (\xi - 1) + x_2 (-\xi - 1) + x_3 (\xi + 1) + x_4 (1 - \xi) \right]$$



# Numerical Integration: Gaussian Quadrature

➤ B matrix:

$$\mathbf{B} = \frac{1}{70 - \xi + 27\eta} \times$$

$$\begin{bmatrix} -4+6\eta-2\xi & 0 & 7-5\eta+2\xi & 0 & 4+5\eta-\xi & 0 & -7-6\eta+\xi & 0 \\ 0 & -6-3\eta+9\xi & 0 & -7-2\eta-9\xi & 0 & 6+2\eta+4\xi & 0 & 7+3\eta-4\xi \\ -6-3\eta+9\xi & -4+6\eta-2\xi & -7-2\eta-9\xi & 7-5\eta+2\xi & 6+2\eta+4\xi & 4+5\eta-\xi & 7+3\eta-4\xi & -7-6\eta+\xi \end{bmatrix}$$



# Numerical Integration: Gaussian Quadrature

► k matrix:

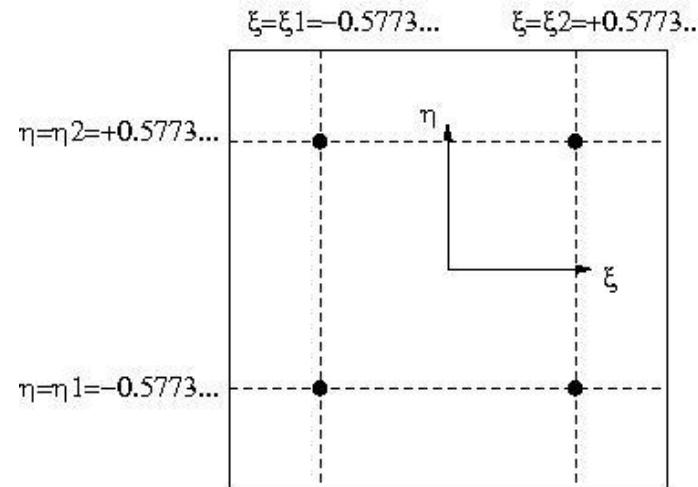
$$\mathbf{E} = \begin{bmatrix} 32000 & 8000 & 0 \\ 8000 & 32000 & 0 \\ 0 & 0 & 12000 \end{bmatrix} \text{ ksi};$$

$$\begin{aligned} \mathbf{k} &= (0.5 \text{ in}) \int_{-1}^1 \int_{-1}^1 \mathbf{B}^T \mathbf{E} \mathbf{B} * J d\xi d\eta \\ &= \int_{-1}^1 \int_{-1}^1 \frac{8}{70-\xi+27\eta} * \underbrace{\begin{bmatrix} 31.25*(236-276\eta-196\xi+315\eta^2-354\eta\xi+275\xi^2) & \dots & 31.25*(70+231\eta-203\xi+90\eta^2-231\eta\xi+43\xi^2) \\ \vdots & \ddots & \vdots \\ \dots & \dots & 31.25*(539+588\eta-490\xi+180\eta^2-228\eta\xi+131\xi^2) \end{bmatrix}}_{\text{sym}} d\xi d\eta \end{aligned}$$

# Numerical Integration: Gaussian Quadrature

➤ 2 x 2 Gauss quadrature:

$$W_i = W_j = 1; \quad i, j = 1, 2.$$



$$\mathbf{k} \approx \sum_{i=1}^2 \sum_{j=1}^2 W_i W_j * [\mathbf{k}'(\xi = \xi_i, \eta = \eta_j)] = [\mathbf{k}'(-\frac{1}{\sqrt{3}}, -\frac{1}{\sqrt{3}})] + [\mathbf{k}'(-\frac{1}{\sqrt{3}}, \frac{1}{\sqrt{3}})] + [\mathbf{k}'(\frac{1}{\sqrt{3}}, -\frac{1}{\sqrt{3}})] + [\mathbf{k}'(\frac{1}{\sqrt{3}}, \frac{1}{\sqrt{3}})]$$

$$\therefore \mathbf{k} \approx \begin{bmatrix} 7028.9 & \dots & 1260.6 \\ \vdots & \ddots & \vdots \\ 1260.6 & \dots & 8489.9 \end{bmatrix} \text{ kips/in.}$$

$$\text{Note: } \mathbf{k}_{exact} = \begin{bmatrix} 7136.6 & \dots & 1263.9 \\ \vdots & \ddots & \vdots \\ 1263.9 & \dots & 8499.0 \end{bmatrix} \text{ kips/in.}$$

➤ Element nodal forces: . . . .

# Numerical Integration: Gaussian Quadrature

## Problem 1

Figure (1) shows a four-node quadrilateral. The (x,y) coordinates of each node are given in the figure. The element displacement vector  $u$  is given as:  $U=[0,0,0.20,0,0.15,0.10,0,0.05]$

Find:

- A- The x-y coordinates of a point  $P$  whose location in the master element is given by  $\xi = 0.5, \eta = 0.5$
- B- The  $u$  and  $v$  displacement of the point  $P$

## Problem 2

Using a 2 by 2 rule evaluate the following integral by Gaussian quadrature, where  $A$  denotes the region shown in Figure (1).

$$\iint_A (x^2 + xy^2) dx dy$$

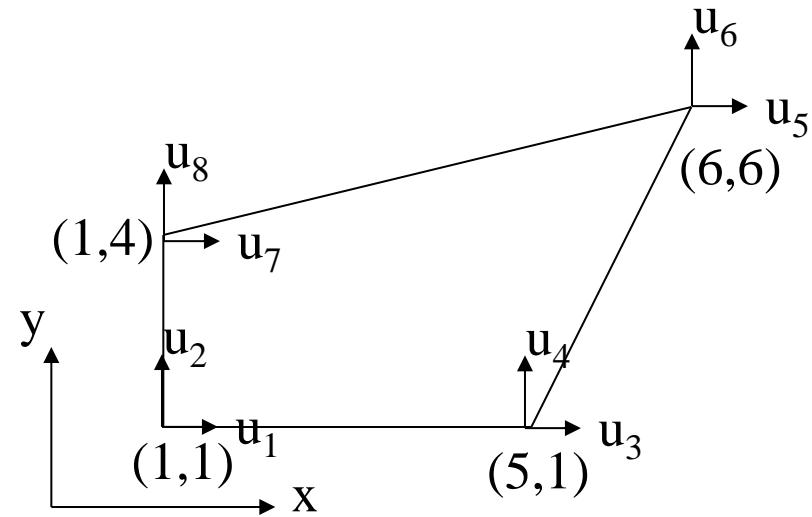


Figure (1)



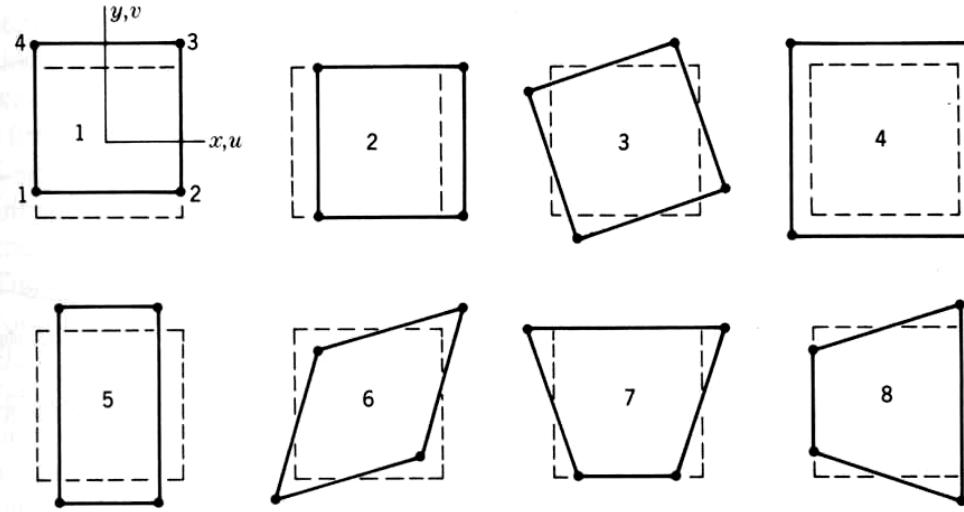
# Numerical Integration: Gaussian Quadrature

## Zero-Energy Modes (Mechanisms; Kinematic Modes) –

- ❖ Instabilities for an element (or group of elements) that produce deformation without any strain energy.
  
- ❖ Typically caused by using an inappropriately low order of Gauss quadrature.
  
- ❖ If present, will dominate the deformation pattern.
  
- ❖ Can occur for all 2D elements except the CST.

## Zero-Energy Modes –

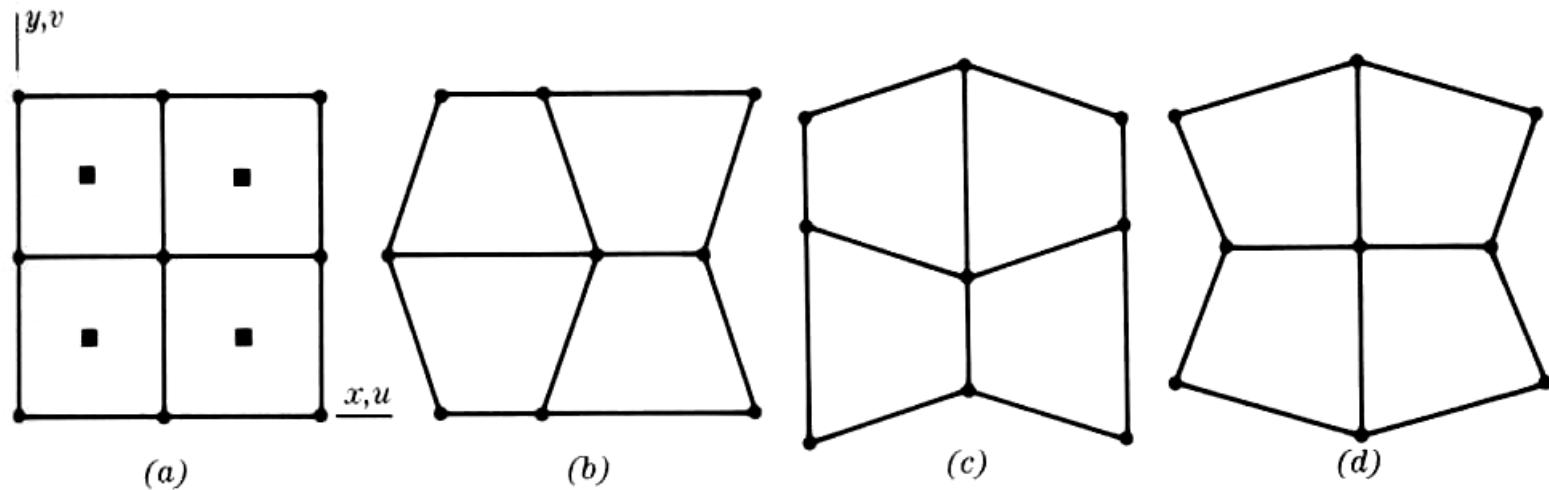
- ❖ Deformation modes for a bilinear quad:



- ❖ #1, #2, #3 = *rigid body modes*; can be eliminated by proper constraints.
- ❖ #4, #5, #6 = *constant strain modes*; always have nonzero strain energy.
- ❖ #7, #8 = *bending modes*; produce zero strain at origin.

## Zero-Energy Modes –

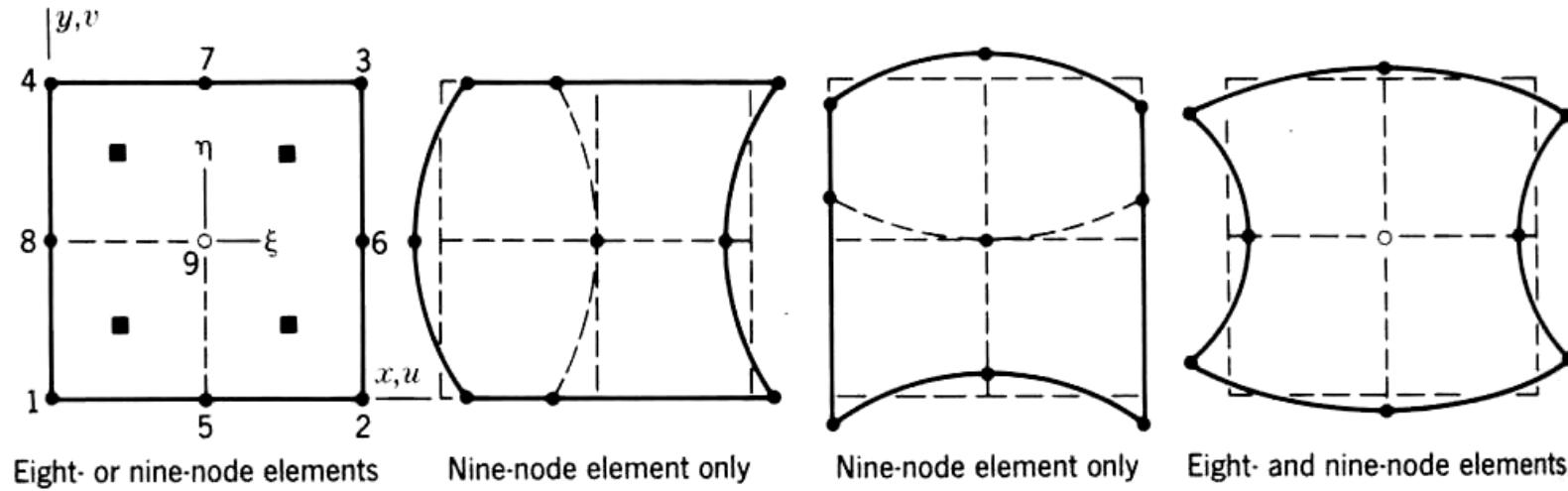
- ❖ Mesh instability for bilinear quads using order 1 quadrature:



“Hourglass modes”

## Zero-Energy Modes –

- ❖ Element instability for quadratic quadrilaterals using 2x2 Gauss quadrature:



“Hourglass modes”

## Zero-Energy Modes –

- ❖ How can you prevent this?
  - ❖ Use higher order Gauss quadrature in formulation.
  - ❖ Can artificially “stiffen” zero-energy modes via penalty functions.
  - ❖ Avoid elements with known instabilities!

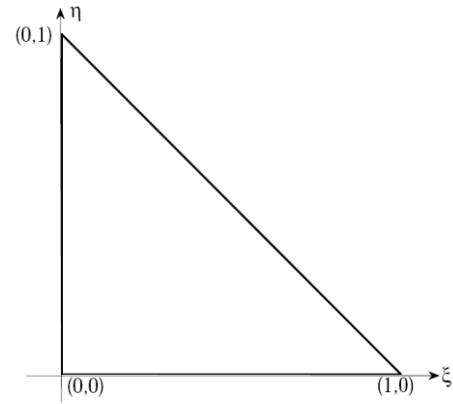
# Gauss Integration for Triangular Region

The gauss points for a triangular region differ from the square region considered earlier. The simplest one is the one-point rule at the centroid with weight  $w_I=1/2$  and  $\xi_1 = \eta_1 = \zeta_1 = 1/3$

$$K^{(e)} = t \int_{\Omega^e} B^T E B \det J d\xi d\eta = \frac{1}{2} \underbrace{t \bar{B}^T \bar{E} \bar{B}}_{\text{Evaluated at Gauss point}} \det \bar{J}$$

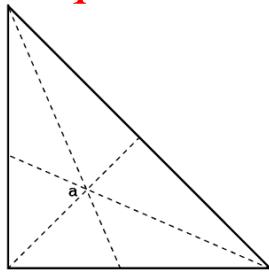
Evaluated at Gauss point

$$I = \int_0^1 \int_0^{1-\xi} f(\xi, \eta) d\eta d\xi \approx \sum_{i=1}^n w_i f(\xi_i, \eta_i)$$

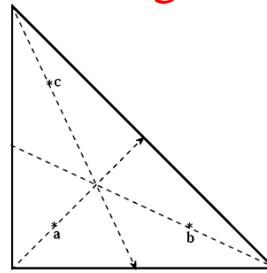


# Gauss Integration for Triangular Region

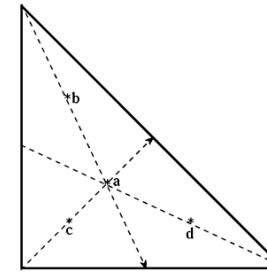
First set of quadrature rules for triangular elements



(a) Linear  
 $a = \left(\frac{1}{3}, \frac{1}{3}\right)$ ,  $w = 1$

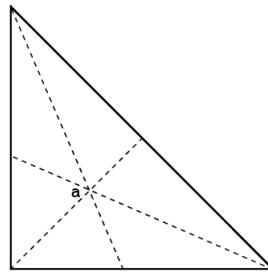


(b) Quadratic  
 $a = \left(\frac{1}{6}, \frac{1}{6}\right)$ ,  $w = \frac{1}{3}$   
 $b = \left(\frac{2}{3}, \frac{1}{6}\right)$ ,  $w = \frac{1}{3}$   
 $c = \left(\frac{1}{6}, \frac{2}{3}\right)$ ,  $w = \frac{1}{3}$

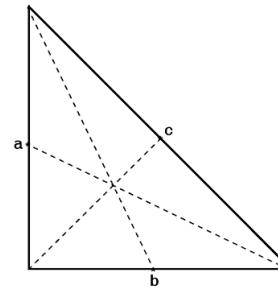


(c) Cubic  
 $a = \left(\frac{1}{3}, \frac{1}{3}\right)$ ,  $w = -\frac{27}{48}$   
 $b = \left(\frac{1}{5}, \frac{3}{5}\right)$ ,  $w = \frac{25}{48}$   
 $c = \left(\frac{1}{5}, \frac{1}{5}\right)$ ,  $w = \frac{25}{48}$   
 $d = \left(\frac{3}{5}, \frac{1}{5}\right)$ ,  $w = \frac{25}{48}$

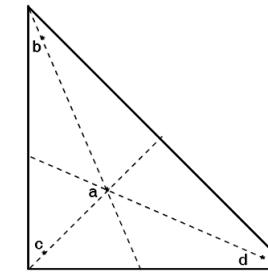
Second set of quadrature rules for triangular elements



(a) Linear  
 $a = \left(\frac{1}{3}, \frac{1}{3}\right)$ ,  $w = 1$



(b) Quadratic  
 $a = \left(0, \frac{1}{2}\right)$ ,  $w = \frac{1}{3}$   
 $b = \left(\frac{1}{2}, 0\right)$ ,  $w = \frac{1}{3}$   
 $c = \left(\frac{1}{2}, \frac{1}{2}\right)$ ,  $w = \frac{1}{3}$



(c) Cubic  
 $a = \left(\frac{1}{3}, \frac{1}{3}\right)$ ,  $w = -\frac{27}{48}$   
 $b = \left(\frac{2}{15}, \frac{11}{15}\right)$ ,  $w = \frac{25}{48}$   
 $c = \left(\frac{2}{15}, \frac{2}{15}\right)$ ,  $w = \frac{25}{48}$   
 $d = \left(\frac{11}{15}, \frac{2}{15}\right)$ ,  $w = \frac{25}{48}$



# Gauss Integration for Triangular Region

$$\int_0^1 \int_0^{1-\xi} f(\xi, \eta) d\eta d\xi \approx \sum_{i=1}^n w_i f(\xi_i, \eta_i)$$

No. of Points (n)	Weight $w_i$	Multiplicity	$\xi_i$	$\eta_i$	$\zeta_i$
One	1/2	1	1/3	1/3	1/3
Three	1/6	3	2/3	1/6	1/6
Three	1/6	3	1/2	1/2	0
Four	-9/32	1	1/3	1/3	1/3
	25/96	3	3/5	1/5	1/5
Six	1/12	6	0.6590276223	0.231933685	0.109039009

Because of triangular symmetry, the Gauss point are occurred in group or *multiplicity* of one, three or six. For multiplicity of three if one Gauss point is at  $(2/3, 1/6, 1/6)$  then the other two Gauss points are located at  $(1/6, 2/3, 1/6)$  and  $(1/6, 1/6, 2/3)$ . For multiplicity of six all six possible permutation of three coordinate are used.